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Approaching the Asymptotic Regime of Rapidly Rotating Convection: Boundary Layers versus Interior Dynamics

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Rapidly rotating Rayleigh-Bénard convection is studied by combining results from direct numerical simulations (DNS), laboratory experiments, and asymptotic modeling. The asymptotic theory is shown to provide a good description of the bulk dynamics at low, but finite Rossby number. However, large deviations from the asymptotically predicted heat transfer scaling are found, with laboratory experiments and DNS consistently yielding much larger Nusselt numbers than expected. These deviations are traced down to dynamically active Ekman boundary layers, which are shown to play an integral part in controlling heat transfer even for Ekman numbers as small as 10^{-7} . By adding an analytical parametrization of the Ekman transport to simulations using stress-free boundary conditions, we demonstrate that the heat transfer jumps from values broadly compatible with the asymptotic theory to states of strongly increased heat transfer, in good quantitative agreement with no-slip DNS and compatible with the experimental data. Finally, similarly to nonrotating convection, we find no single scaling behavior, but instead that multiple well-defined dynamical regimes exist in rapidly rotating convection systems.

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Rapidly rotating thermal convection is ubiquitous in nature. It occurs in the ocean, in the liquid metal cores of terrestrial planets, in gas giants, and in rapidly rotating stars. All these systems are highly turbulent, but at the same time Coriolis forces chiefly control their dynamics. It is this dominating role of Coriolis forces which gives convection in many large-scale natural systems its distinctive character.

Different from nonrotating convection, where large regions of the parameter space have been explored extensively over the last decades [1], both experiments and direct numerical simulations (DNS) face serious difficulties in entering the turbulent, but rotationally constrained regime. While experiments easily reach high levels of turbulence, they struggle to ensure that Coriolis forces remain dominant in the force balance [2–4]. Numerical simulations suffer from the enormous range of spatial and temporal scales that need to be resolved. As a consequence, the available data are scarce, and the scaling laws that are needed for quantifying the effects of convection in large-scale natural systems remain poorly constrained.

The canonical framework to study rotating convection is the rotating Rayleigh-Bénard system. A plane fluid layer of depth *H*, destabilized by a constant temperature difference ΔT between the boundaries, rotates about a vertical axis with angular velocity Ω . Within the Boussinesq approximation, three nondimensional parameters control the system behavior. The Rayleigh number Ra = $g\alpha\Delta TH^3/\kappa\nu$, where *g* denotes gravitational acceleration, α the thermal expansion coefficient, ν the kinematic viscosity, and κ the thermal diffusivity, measures the forcing strength. The Ekman number $E = \nu/2\Omega H^2$ is defined as the ratio of the rotational time scale to the viscous diffusion time scale. Finally, the Prandtl number $Pr = \nu/\kappa$ signifies the efficiency of viscous relative to thermal diffusion. A combination of these parameters, the convective Rossby number $Ro_c = \sqrt{Ra/PrE}$, is often used as a proxy for the importance of rotation relative to the thermal forcing.

Rapidly rotating convection is characterized by $\text{Ro}_c \ll 1$, with the asymptotic limit $\text{Ro}_c \rightarrow 0$ representing an important limiting case. Previous theoretical work [5,6] has shown that the governing equations can be simplified substantially in this limit. The resulting reduced set of equations, essentially a nonhydrostatic quasigeostrophic model, is expected to hold for $\text{Ra} \leq O(E^{-5/3})$ and $\text{Pr} \geq O(E^{1/4})$ [7]. The Ekman layer is assumed to become passive at small E [8,9], and its $O(E^{1/2})$ vertical transport is assumed to be negligible for the dynamics in this regime [6,10,11]. Numerical simulations using the reduced equations have revealed a rich dynamical behavior [6,7,11], characterized by the existence of several distinct flow regimes, each associated with different heat transfer properties.

Previous experiments [12–15] and DNS [15,16] barely reach into the rapidly rotating regime, and their relation to the asymptotic model is thus unclear. Here, we attempt to establish such a relation by pushing direct numerical simulations and laboratory experiments further into the rapidly rotating regime, where a comparison to asymptotic models becomes meaningful. A central result of our study is that the behavior in the *bulk* is well described by the asymptotic model, while, contrary to common expectations, the viscous boundary layer dynamics largely controls the heat transfer scaling whenever the fluid layer is confined between no-slip boundaries. In fact, even for the lowest Ekman numbers that can be reached in laboratory experiments and DNS today, viscous boundary layers are shown to massively boost the heat transfer in the low Rossby number regime, leading to a considerable increase of the exponent α in power laws of the form Nu = f(Pr, E) Ra^{α}, where the Nusselt number Nu is defined as the total heat flow normalized by the purely conductive value.

To illustrate this, Fig. 1 shows the Nusselt numbers found in DNS and in experiments at $E \approx 10^{-7}$. In order to compare our results with the asymptotic predictions, we use $\widetilde{\text{Ra}} = \text{Ra}E^{4/3}$ as a measure for the supercriticality of the system [6]. The DNS employ both no-slip and stress-free boundary conditions, and are carried out in a horizontally periodic box with an aspect ratio $\Gamma = 10l_{\perp}H^{-1}$, where l_{\perp} is the critical wavelength of the marginally unstable mode for the stress-free case. Resolutions up to $576 \times 576 \times 513$ grid points are used to fully resolve the flow down to the dissipation scale. Care has also been taken to resolve the thin Ekman layers, characterized by an $O(E^{1/2}H)$ thickness, with a minimum of ten grid points. Experiments made



FIG. 1 (color online). Nusselt number and interior temperature gradient at the midplane as a function of $\widetilde{\text{Ra}}$. The filled symbols show DNS results at $E = 10^{-7}$, while the open symbols are asymptotic predictions [7]. The stars show laboratory values obtained in a cylindrical rotating water tank with $5.79 \le \text{Pr} \le 6.2$ and $10^{-7} \le E \le 1.1 \times 10^{-7}$.

using water with $Pr \approx 7$ as the working fluid are carried out in axially aligned cylindrical containers with diameters of 18.73 cm and heights of 40, 80, and 160 cm. A detailed description of the numerical and experimental techniques are, respectively, given in [17] and [2,4].

For stress-free boundary conditions and moderate Ra, Fig. 1 reveals an excellent agreement between DNS and the asymptotic results, which becomes poorer for Ra > 50. The results are broadly compatible with $\alpha = 3/2$ at high Ra, as expected for diffusion independent turbulent heat transfer [11]; see also [18]. The interior temperature gradient found in the DNS follows the asymptotic results closely over the entire range of Ra considered in this study. In contrast, DNS and the asymptotic results diverge strongly for no-slip boundary conditions. From onset on, the DNS reveals a much steeper heat transfer scaling, broadly compatible with the $\alpha \sim 3$ scaling proposed in [2], which flattens out as soon as the interior temperature gradient reaches its minimum. At this point, the different scaling behavior has resulted in a Nusselt number that exceeds the corresponding values obtained for stress-free boundaries by more than 800% for Pr = 7. Data from laboratory experiments, denoted by stars in Fig. 1, are in line with these numerical findings.

The excellent agreement found between DNS with stress-free boundaries and the asymptotic results suggests that the bulk dynamics is well captured by the asymptotic model. This conjecture is supported by the fact that the reduced model also provides a qualitatively correct picture of the dominant flow regimes in the bulk of the convective layer for no-slip DNS. For small Ra, a cellular regime is found [Fig. 2(a)], which gives way to a regime characterized by so-called convective Taylor columns (CTCs) [19] as Ra is increased [Fig. 2(b)]. The up- and downwellings form intense columnar structures, which are less densely packed than in the cellular regime. Their intense vortex cores are shielded by weaker sheets of opposite vorticity and temperature anomaly, preventing them from being destroyed by vortex-vortex interactions. The interior temperature gradient decreases with Ra through the cellular and CTC regime. Its downward trend is finally broken when the CTCs begin to lose their shields, which exposes the vortex cores. Violent vortical interactions then destroy the vertically coherent columns and lead to the formation of plume-like structures [Fig. 2(c)], which is accompanied by an increasing interior temperature gradient. For Pr = 1, the system directly evolves from the cellular regime to this plume state, without forming CTCs. A further increase of Ra then leads to a complete breakdown of vertical coherence for Pr = 1 [Fig. 2(d)], the interior temperature gradient saturates and the flow enters a regime called geostrophic turbulence (GT). Similar regimes are also observed in the stress-free case, with the regime boundaries agreeing well with the asymptotic model even quantitatively.



FIG. 2 (color online). Thermal anomaly $\theta = T - \overline{T}$ in DNS at $E = 10^{-7}$ with no-slip boundary conditions. (a) Cellular regime, $\widetilde{Ra} = 10$, Pr = 1, (b) convective Taylor columns, $\widetilde{Ra} = 25$, Pr = 7, (c) plumes, $\widetilde{Ra} = 70$, Pr = 7, and (d) geostrophic turbulence, $\widetilde{Ra} = 90$, Pr = 1. For better visibility, the domain has been stretched horizontally by a factor of 4.5. θ is scaled with ΔT in all cases.

An interesting feature of the GT regime is the formation of large-scale barotropic vortices driven by an upscale transport of kinetic energy [7,20]. The effect is most clearly observed in the stress-free case (Fig 3). The formation of large scale vortices in rotating convection has recently been observed in simulations at larger E [21,22], where the large-scale energy accumulates predominantly in cyclonic structures. Such symmetry breaking is predicted to be absent in the asymptotic small Rossby number case [20]. Indeed, our DNS reveals the generation of both, strong cyclonic and anticyclonic vortices, which shows that symmetry slowly tends to be restored with decreasing Rossby number. The formation of large scale coherent vortices is inhibited in DNS that employ no-slip boundaries, similar to the results of [23].

Our results thus suggest that at $E = 10^{-7}$, the bulk dynamics shows clear signs of asymptotic behavior, while viscous boundary layers strongly increase the heat transfer efficiency, leading to a different scaling behavior than predicted asymptotically. The mechanical boundary conditions thus exert a controlling influence on the heat transfer scaling, a result that is unexpected for the low Ekman number considered here. Studies of the linear problem [8] suggest that the onset of convection becomes largely independent of the mechanical boundary conditions for $E \leq 10^{-6}$. While nonlinear contributions from Ekman pumping are known to increase the heat transport at



FIG. 3 (color online). Formation of large-scale, barotropic vortices of both signs of vorticity in the GT regime for stress-free boundaries at $E = 10^{-7}$, $\widetilde{\text{Ra}} = 90$, Pr = 1. Shown is (a) vertical vorticity, (b) horizontal kinetic energy, (c) temperature anomaly at z = 0.99, and (d) vertically averaged vertical vorticity. Units are κ/H^2 , κ^2/H^2 , ΔT , and κ/H^2 , respectively.

moderate Ekman numbers [24], it is generally expected that this effect becomes small in the low *E* regime, as both the Ekman layer thickness and the associated secondary flow should decrease with $O(E^{1/2})$, such that the Ekman layer finally becomes passive [6,10,11]. It is remarkable that even at $E = 10^{-7}$, where the Ekman layer covers only about 0.1% of the layer depth, an increase by almost an order of magnitude in the heat transfer is observed.

The effects of no-slip boundaries can be modeled using boundary conditions that parametrize fluid pumping into and out of the linear Ekman layers [25]. Pumping is added to otherwise stress-free boundary conditions by enforcing

$$\frac{\partial u_x}{\partial z} = \frac{\partial u_y}{\partial z} = 0, \qquad u_z = \pm \left(\frac{E}{2}\right)^{1/2} \omega_z H \quad \text{at } z = 0, H$$
(1)

in DNS, where ω_z denotes vertical vorticity. The plus sign applies to the lower and the minus sign to the upper boundary, respectively. Note that this choice isolates the Ekman pumping effects, while neglecting the viscous dissipation in the boundary layer that plays a central role in theories of nonrotating convection [26]. Figure 4 shows the results obtained. Both the heat transfer and the interior temperature gradients found in the DNS are well captured by the pumping parametrization [Figs. 4(a), 4(b)]. Panels 4(c) and 4(d) show profiles of the rms horizontal and vertical velocity and of conductive and advective heat transfer for the special case Pr = 7, Ra = 20. The profiles line up well within the bulk, and reveal a substantial



FIG. 4 (color online). Results obtained with parametrized Ekman pumping. Filled symbols denote no-slip cases, while open symbols show results obtained by applying the pumping boundary conditions (1) to the full equations. Vertical profiles for $E = 10^{-7}$, Pr = 7, $\tilde{Ra} = 20$ are shown in (c), with a boundary layer blowup in (d) for no-slip (solid) and Ekman pumping (dashed) boundary conditions. (u_{\perp} : horizontal velocity, *w*: vertical velocity, *T*: temperature, given in units of κ/H and ΔT , respectively).

advective heat transport all the way to the edge of the Ekman layer. The steepest $Nu(\widetilde{Ra})$ scaling is observed in the CTC regime with Pr = 7, where the Ekman flow at both boundaries acts to efficiently increase the heat transport through these vertically coherent structures. For Pr = 1, where plumes develop directly from the cellular regime as \widetilde{Ra} is increased, Ekman pumping has a much smaller effect on the heat transfer.

For clarity, the presentation so far has focused exclusively on cases with $E = 10^{-7}$. Figure 5 illustrates the system behavior over the range $10^{-5} \le E \le 2.5 \times 10^{-8}$ found in no-slip DNS and laboratory experiments. Instead of using $\tilde{Ra} = RaE^{3/4}$, here we normalize Ra by the critical value for the onset of convection Ra_c as determined from Chandrasekhar's "first approximation" [27,28], because the asymptotic scaling law $Ra_c = 8.6956E^{-4/3}$ becomes inaccurate at larger *E*.

Surprisingly, in the presence of no-slip boundaries, no obvious convergence towards the asymptotic predictions is observed with decreasing Ekman number. For Pr = 7, the steep scaling regime extends to higher \tilde{R} at lower E, such that the maximum deviations from the asymptotic theory in fact *increase* with decreasing E. This trend is observed in both DNS and experiments and likely occurs because vertically coherent structures remain prevalent over a larger supercriticality range at lower E. For Pr = 1, the effect is



FIG. 5 (color online). Ekman number dependence of Nu and pumping efficiency. Colored symbols represent no-slip DNS results (full symbols: Pr = 1, open symbols: Pr = 7), gray symbols show the asymptotic results. The panel in the lower left also contains experimental data (+: $2.56 \times 10^{-8} \le E \le 2.77 \times 10^{-8}$, *: $0.93 \times 10^{-7} \le E \le 1.15 \times 10^{-7}$, ×: $0.8 \times 10^{-5} \le E \le 1.5 \times 10^{-5}$, with $5 \le Pr \le 11$ in all cases.). The right panels show the rms vertical velocity close to the edge of the Ekman layer, measured at $z_E = 3/4\pi\sqrt{2E}$, normalized by the rms vertical velocity at midlayer, a ratio we call *S* here. Note that at $z = z_E$, classical Ekman layer theory predicts the pumping velocity to be identical to its far-field value, such that values at this location provide a good measure of Ekman pumping.

less pronounced, which is in accord with the fact that plume structures develop at much smaller \tilde{R} and mix into the interior before reaching the opposite boundary, thus diminishing the effects of pumping. A close inspection of the numerical data shown in Fig. 5 suggests that there might be a weak trend towards the asymptotic modeling results for small $\tilde{R} \leq 2$ at both Pr = 1 and Pr = 7, but data extending to much lower *E*, at present a significant challenge in the DNS and laboratory setting, are necessary to substantiate this effect.

The panels on the right of Fig. 5 show the relative strength of the Ekman flow in the simulations, measured by a quantity *S* that we define as the rms vertical velocity at $z = 3/4\pi\sqrt{2E}$ (i.e., close to the edge of the Ekman layer) normalized by its value at midlayer. The Ekman flow reaches amplitudes roughly between ten and forty percent of the vertical velocity in the bulk, and thus clearly is not negligible. The quantity *S* increases with \tilde{R} and decreases with *E*, for moderate \tilde{R} roughly proportional to $E^{1/6}$, as shown in the lower right panel of Fig. 5, where the data cluster in two groups depending on Pr. The $E^{1/6}$ scaling is expected from (1) if we employ the usual estimates $[L] = O(E^{1/3}H)$ and $[u] = O(E^{-1/3}\nu/H)$ for the typical horizontal length scale and flow velocities [7].

Our results show that in the studied parameter range, the Nu(Ra) scaling is not converging towards the asymptotic modeling results even though the Ekman flow substantially decreases with decreasing E. This finding, and the enormous magnitude by which pumping increases the heat transfer need to be explained by future theory. A positive feedback between thermal boundary layer instabilities and the secondary Ekman flow supporting them may be considered in this context. It clearly remains to be seen if pumping ultimately loses its significance in the very low, but finite Rossby number regime, as presently presumed by theory. Currently, neither experiments nor simulations support this claim. A promising avenue towards gaining a better understanding is the inclusion of parametrized Ekman pumping into the asymptotic theory, as this approach works in our full simulations. Equally important are experiments and DNS reaching still lower E values. Even if pumping effects should finally become negligible, Fig. 5 suggests that values several orders of magnitude smaller than $E = 10^{-8}$ are needed in order to observe this.

Another finding of this study is that, similar to the situation in nonrotating convection, heat transport in the low Rossby number regime cannot be characterized by a single, universal scaling exponent α . Instead, different dynamical regimes exist, all exhibiting their own heat transfer characteristics, as also suggested by asymptotic theory [7,11]. The steep $\alpha \sim 3$ scaling, for example, breaks down well within the low Rossby number regime (Ro_c ≈ 0.01 for Pr = 7, $E = 10^{-7}$) when CTCs evolve into plumes. Beyond this transition, a much slower growth of Nu with \tilde{R} is observed. Unfortunately, laboratory experiments have yet to reach small enough \tilde{R} at low E to investigate this regime change [2–4]. The experimental data presented here are no exception (cf. Fig. 5).

The usual motivation for studying rotating convection is its ubiquitous occurrence in geo- and astrophysical systems. Our findings suggest that Ekman pumping, which is typically regarded as negligible in this context [29,30], may play a more prominent role than previously thought.

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